

# Control Systems

(Prof. Casella)

Written Exam – July 2<sup>nd</sup>, 2015

Surname:.....

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Reg. Number:.....

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## Notices:

- This booklet is comprised of 7 sheets – Check that it is complete and fill in the cover.
- Write your answers in the blank spaces with short arguments, including only the main steps in the derivation of the results.
- You are not allowed to leave the classroom unless you hand in the exam paper or withdraw from the exam.
- You are not allowed to consult books or lecture notes of any kind.
- Please hand in only this booklet at the end of the exam – no loose sheets.
- The clarity and order of your answers will influence how your exam is graded.

**Question 1**

With reference to feed-back control systems, explain why the input-output dynamic behaviour of the plant to be controlled is crucial for the system performance. Also explain why, on the other hand, a very precise knowledge of that dynamic behaviour is not required, if the controller is designed properly.

**Question 2**

Briefly explain why the stability of a linear system with transfer function  $G(s)$  is completely determined by its poles. Then, state precisely under which conditions such system will be asymptotically stable, simply stable, or unstable.

### Question 3

Consider a one-dimensional spring-mass mechanical system, in which a body of mass  $M$  is connected to the ground by a spring and by a damper, and subject to an external force  $F$ . Contrary to a standard spring-mass harmonic oscillator, the spring has a nonlinear force-displacement characteristic, where the force grows more than proportionally to the spring deformation.

The equations describing the system are the following, where  $x$  is the body displacement,  $v$  is the body velocity,  $F_s$  is the spring force,  $F_d$  is the damper force,  $K$  and  $x_0$  are the spring parameters, and  $h$  is the damper friction coefficient.

$$M \ddot{x} = F_s + F_d + F$$

$$F_s = -\frac{K}{x_0^2} x (x^2 + x_0^2)$$

$$F_d = -hv$$

**3.1** Write down the state and output equations in standard state-space form, considering  $F$  as input and  $x$ ,  $v$  as outputs

**3.2** Compute the equilibrium conditions for the system

**3.3** Write down the system's linearized equations around a generic equilibrium

**3.4** Assuming the equilibrium value of  $F$  is zero, compute the transfer functions of the system between the deviations of the input  $\Delta F$  and the deviations of the outputs  $\Delta x$  and  $\Delta v$ .

**3.5** Determine under which conditions the system shows an oscillatory behaviour in response to a unit step change of the input  $\Delta F$ , then plot the qualitative diagrams of the corresponding  $\Delta x(t)$  and  $\Delta v(t)$ .

**3.6** Assume now that the equilibrium value of  $F$  is no longer zero, but rather that the spring is pre-loaded by a suitable force value such that the corresponding equilibrium displacement is  $x_0$ . Explain how the plots determined at the previous point change, all other parameters being unchanged.

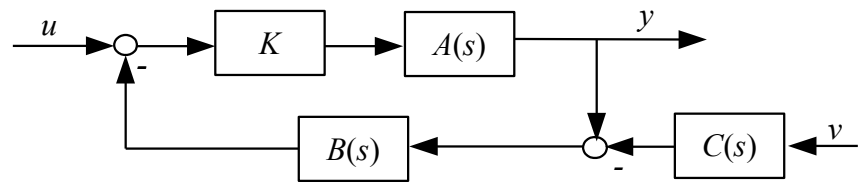
### Question 4

Considering the following block diagram

$$A(s) = \frac{10}{1+s}$$

$$B(s) = \frac{1}{1+s}$$

$$C(s) = \frac{5}{1+s}$$



4.1 Compute the transfer functions between the inputs  $u$  and  $v$  and the output  $y$ .

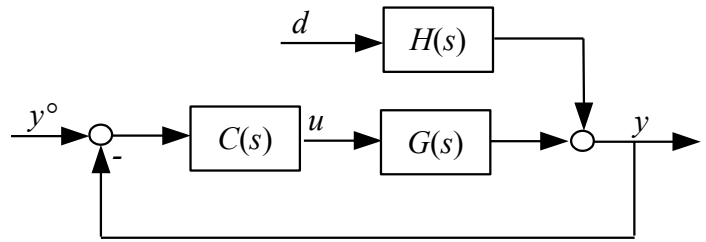
4.2 Determine for which values of the parameter  $K$  the system shows exponentially diverging oscillations in response to step changes of the inputs.

### Question 5

Consider the following control system, where the unit of time constants is the second:

$$G(s) = \frac{500}{s(1+100s)(1+30s)(1+10s)(1+5s)}$$

$$H(s) = \frac{10}{s(1+100s)}$$



**5.1** Design a PI or PID controller with a bandwidth of 0.003 rad/s and at least  $60^\circ$  phase margin.

**5.2** Design a PID controller with the highest bandwidth you can get for a phase margin of  $45^\circ$ , assuming  $N = 10$  for the derivative action.

**5.3** Plot the qualitative diagrams of the response of the controlled output  $y$  to a step change in the disturbance  $d$  for the two controllers designed at points 5.1 and 5.2.

**5.4** Determine the asymptotic amplitude of the oscillations of the manipulated variable  $u$  corresponding to a disturbance  $d = \sin(2t)$  for the two controllers designed at points 5.1 and 5.2.